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Dipartimento di
Ingegneria Industriale



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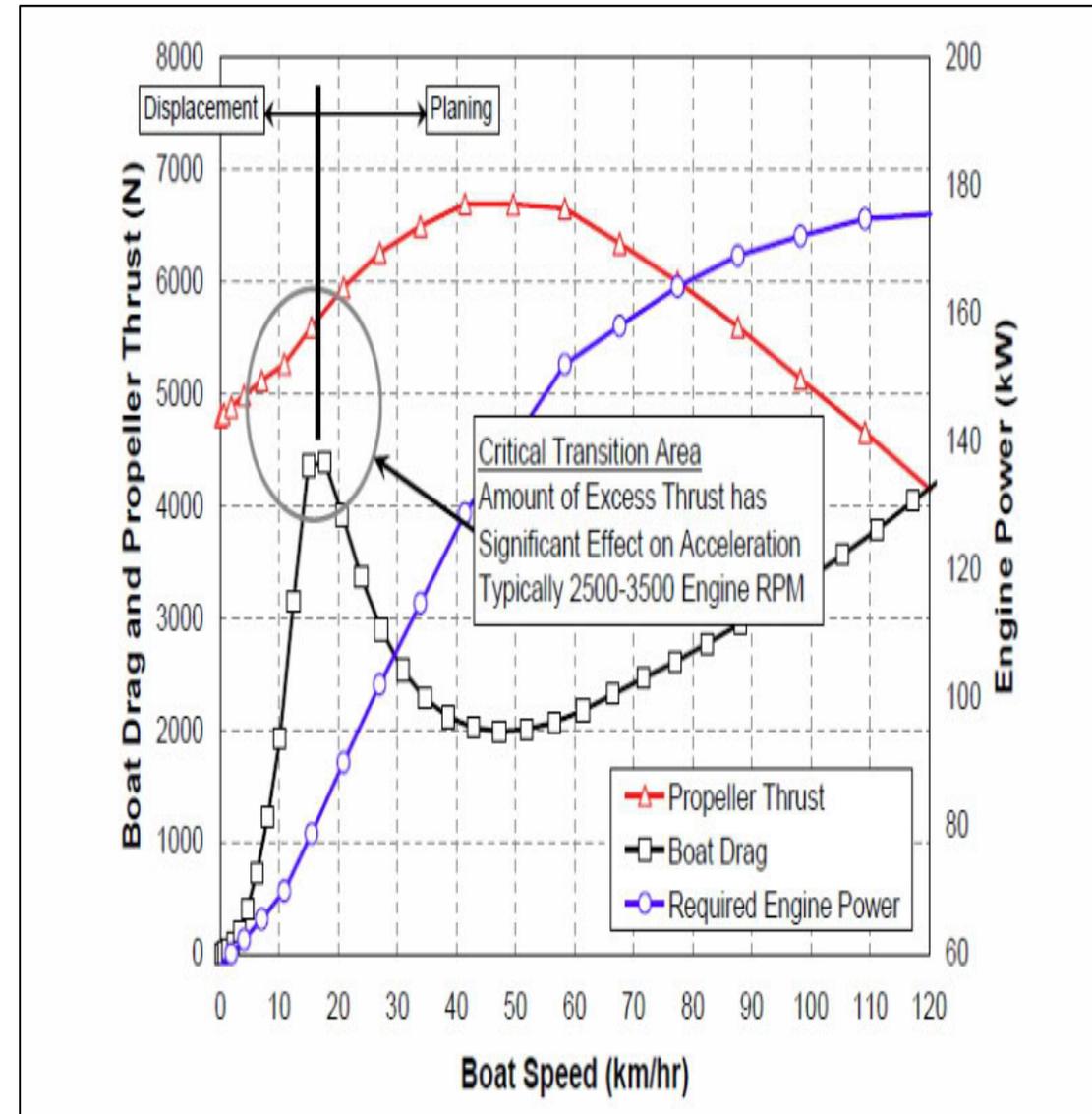
*Leonardo - Engineers for Integration
Torino (Italy)*

TURBOCHARGHED DIESEL ENGINE COUPLED WITH PLANING VESSEL

Ricardo Software 2015 European User Conference

Field of research

- Engines for planing boats
 - ✓ **Marine turbocharged Medium Duty diesel engines**
 - Matching of engine - propeller - hull
 - ✓ **Two competing requirements:**
 - High power density in order to reach high peak velocity;
 - Low End Torque Availability to overcome the peak of resistance in the displacement to planing transition phase.



Framework: Boat Displacement to Planing transition

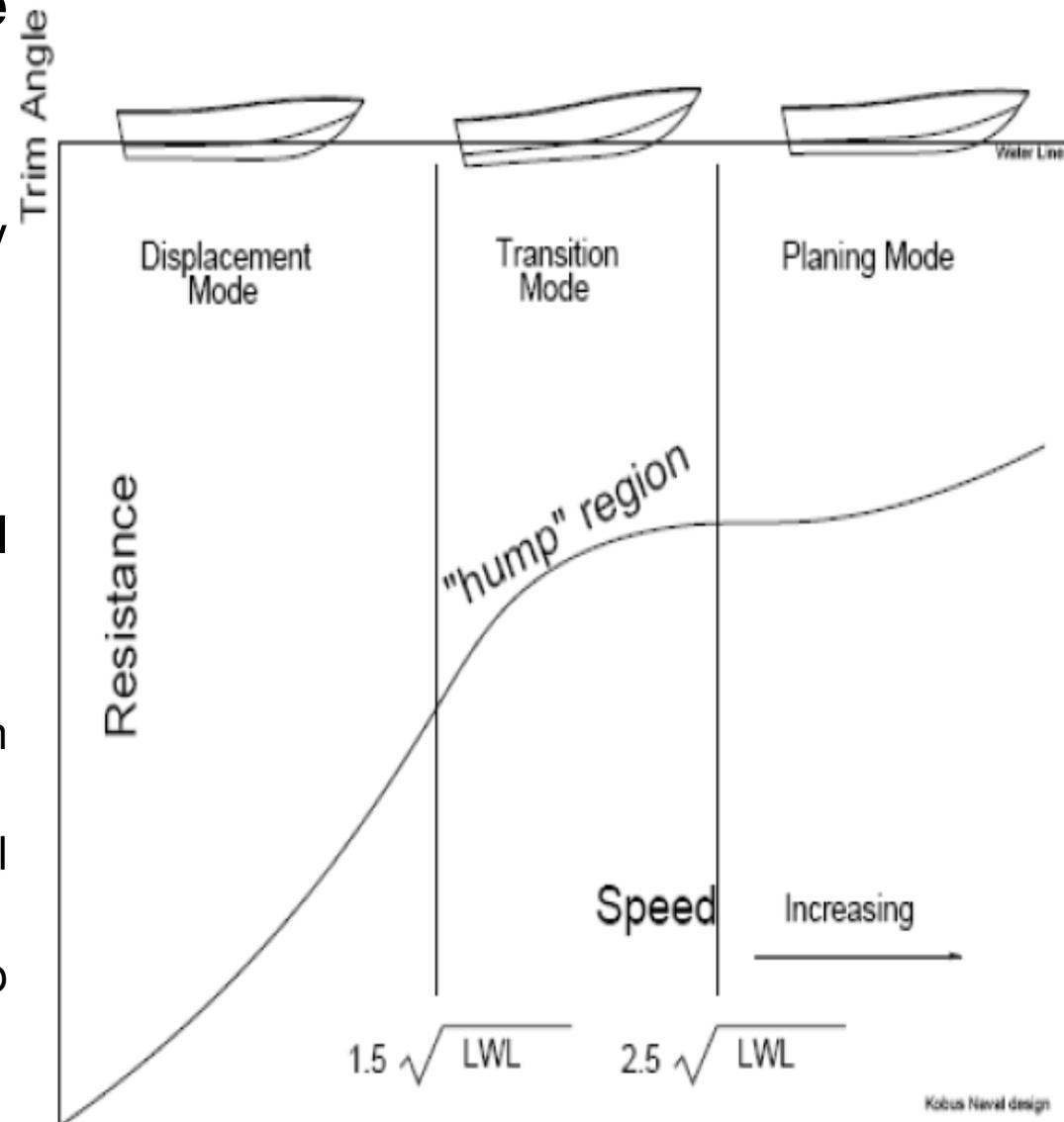
➤ Resistance to the motion of the planing boat:

✓ *Displacement condition*

- The resistance grows with the velocity up to the *critical transition velocity*

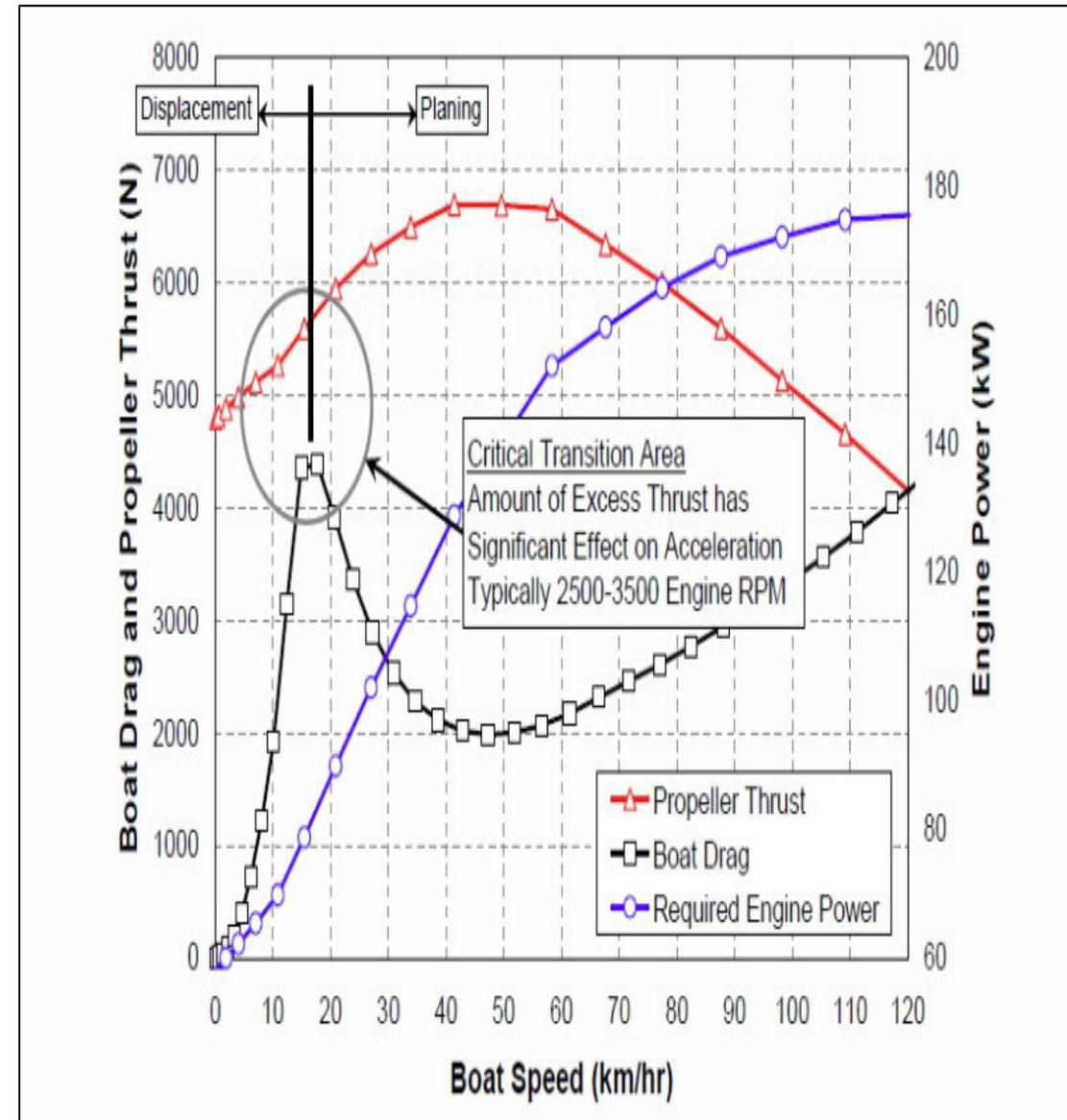
✓ *Planing condition*

- The hull shape allows the partial raising of the boat on the water
 - Hydrodynamic reaction
 - Reduction of the viscous force (reduction of the wet surface)
 - Strong reduction of the residual resistance (wave resistance + vortices)
 - The total resistance gradually grows up to the maximum velocity



Engine Matching to Transmission

- Turbocharger Matching to boat:
 - ✓ Surplus of power after transition:
 - Standard solution
 - *Dual gear transmission*
 - *Variable pitch propellers*
 - Innovative solutions
 - *Electrical or mechanical charger in series with the turbocharger*
 - *MGU Turbocharger*
 - » No turbo-lag and overpressure air immediately available to the engine even at low rpm
 - » It is possible to use the turbocharger to recover the surplus energy when the peak of resistance is overtaken
 - » The surplus energy used for the subsequent acceleration or for the auxiliary equipment



Aim of the activity:

Provide a simulation tool for planing boat transmission dynamics

✓ Development of an *engine-transmission-boat* numerical model:

○ *Goal:*

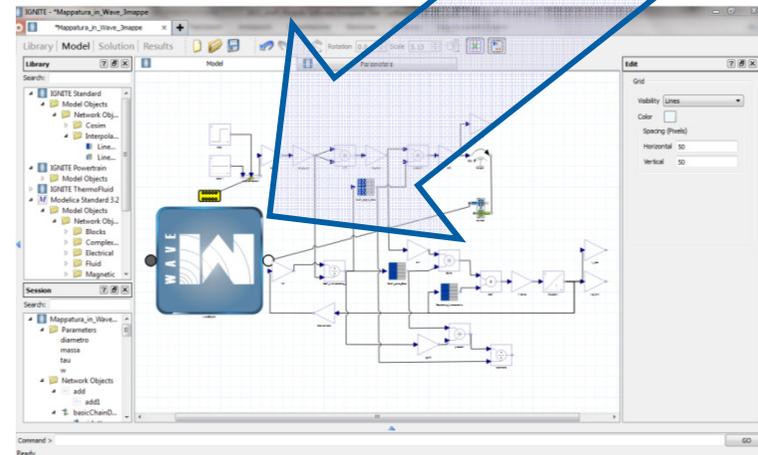
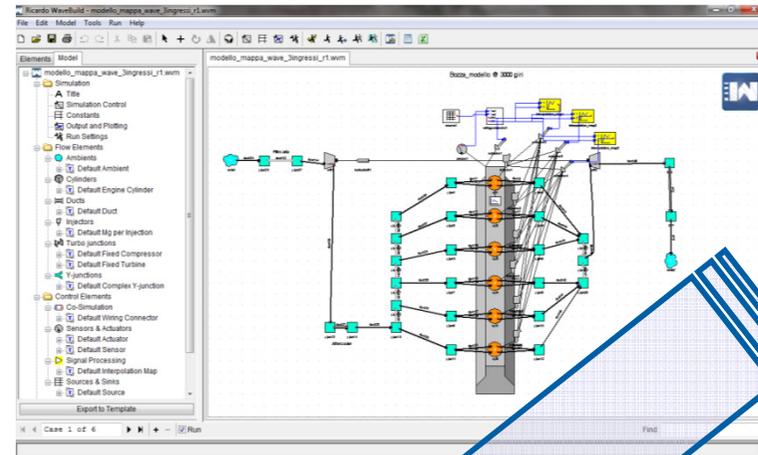
- evaluate during the design phase if the **engine-turbocharger matching** is able to guarantee both **high power** and **low end torque to start “planing”**

○ *Numerical Tools:*

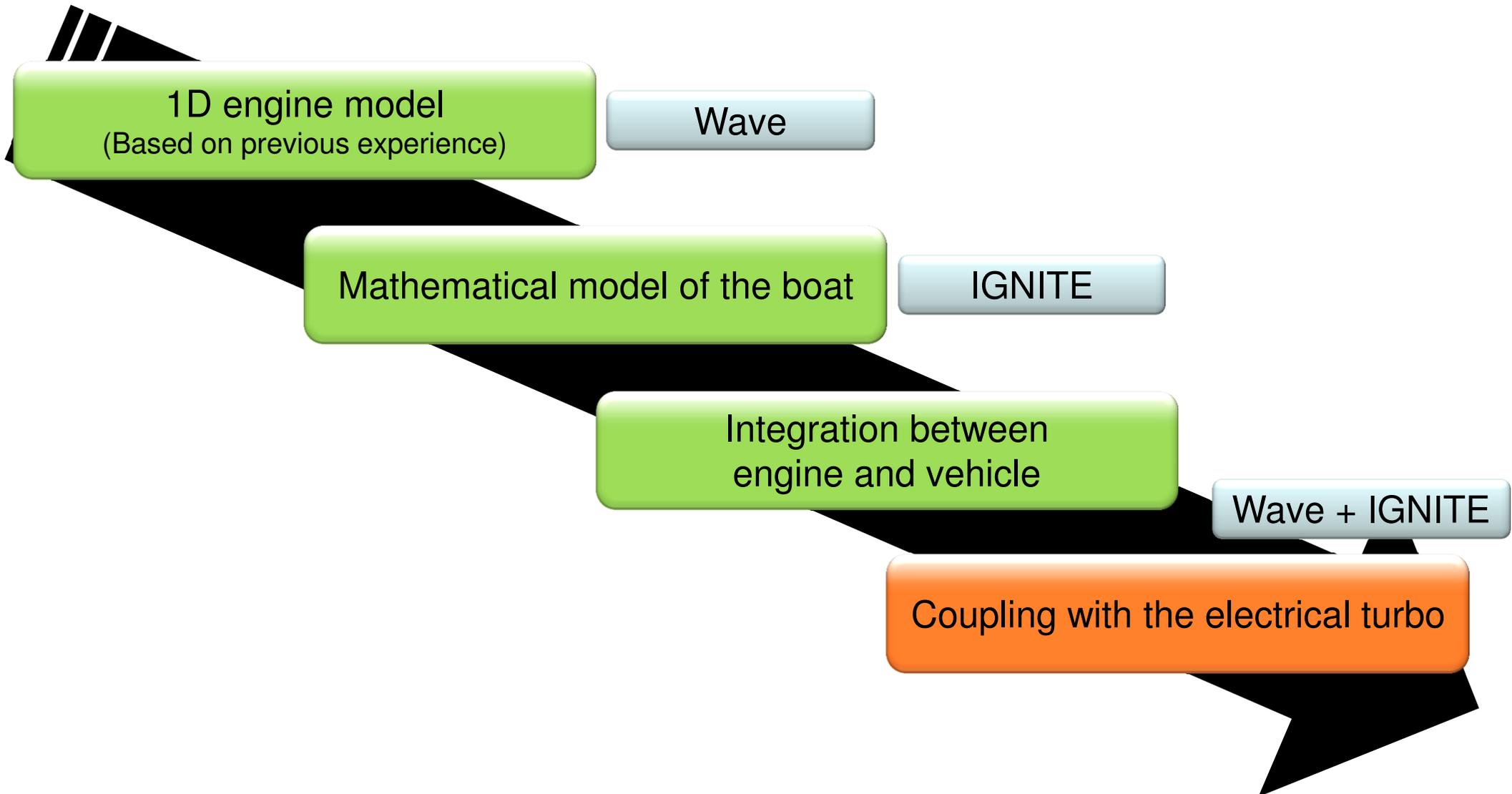
- 1D engine model built in **Wave**
- Mathematical model to simulate the longitudinal dynamic of the boat implemented in **IGNITE**
- Integration of the engine model with the boat model:

○ *Future development:*

- evaluate the coupling among engine – MGU Turbocharger, operating also in energy recovery mode.



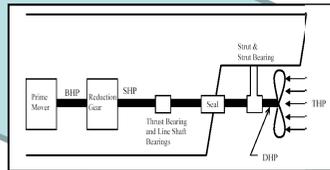
Road Map



Boat dynamic

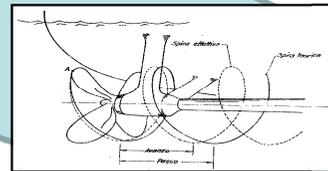


Engine torque
Engine rpm



Propeller torque
Propeller rpm

Propeller coefficients



Boat drag
Propeller thrust

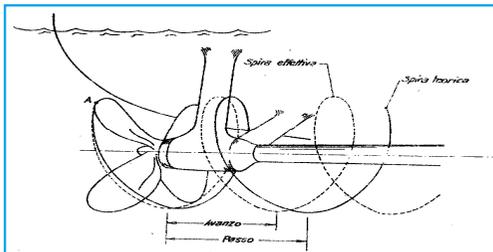
Speed of advance

Resistance



Vehicle speed

Boat dynamic – mathematical model



PROPELLER

➤ Simplest Propeller Model:

- ✓ only thrust to the hull transmitted to boat
- ✓ Immersed free propeller
 - Characteristics independent from the trim of the boat (e.g. from the flotation or boat pitch angle)
 - Characteristic parameters of the propellers
 - Grouped by reference series
 - » Number of blades
 - » Pitch / Diameter Ratio
 - » Projected blade area / Disc area Ratio

Ref.: Guidance and control of ocean vehicles, T. Fossen

Ref.: Nozioni di macchine marine, Accademia Navale

Ref.: Fondamenti di Idrodinamica, U. Costaguta

Ref.: Marine propellers and propulsion, J. Carlton

Ref.: The prediction of power performance of planing crafts, J.B. Hadler, SNAME, 1966

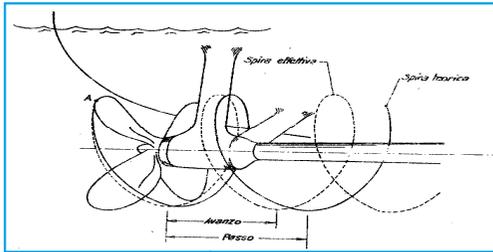
Ref.: Fondamenti e applicazioni di propulsione navale, G. Trincas, Università di Trieste

Ref.: Lo scafo da diporto, A. Payer, M. Vassalle

Ref.: Le turbomacchine, G. Manfrida, S. Stecco

Ref.: Meccatronica, Elementi di trazione elettrica, B. Allotta, L. Pugi

Boat dynamic – mathematical model



PROPELLER

- Performance of the propeller from experimental tests in open-water available in literature

- *Advance coefficient*
- *Thrust coefficient*
- *Torque coefficient*

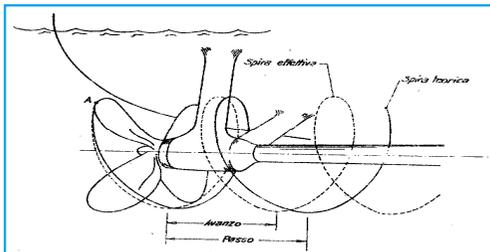
$$J = \frac{v_a}{n d}$$

$$K_T = \frac{T}{\rho n^2 d^4}$$

$$K_Q = \frac{Q}{\rho n^2 d^5}$$

- v_a Propeller's speed of advance
- n Propeller's rotational speed
- d Propeller's diameter
- ρ Water density
- T Thrust
- Q Propeller's torque

Boat dynamic – mathematical model



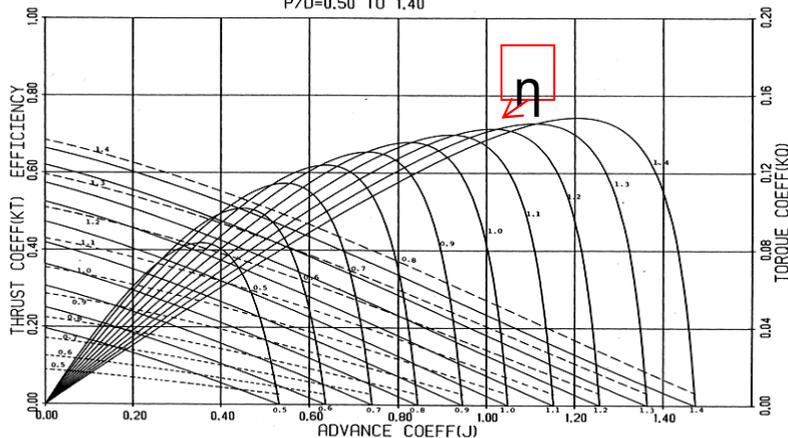
PROPELLER

- Propeller efficiency (η)
 - ✓ Function of the propeller coefficients

$$\eta = \frac{T v_a}{2\pi n Q} = \frac{P_T}{P_D} = \frac{K_T \rho n^2 d^4 v_a}{2\pi K_Q \rho n^3 d^5} = \frac{K_T v_a}{K_Q 2\pi n d} = \frac{K_T J}{2\pi K_Q}$$

- P_T : Thrust power of the propeller
- P_D : Power absorbed by the propeller behind the hull

FIGURE 43. WAGENINGEN B-SERIES PROPELLERS
FOR 4 BLADES $AE/AO = 0.800$
 $P/D = 0.50$ TO 1.40



$K_T (J) \rightarrow$ continuous line
 $K_Q (J) \rightarrow$ dashed line

Boat dynamic – mathematical model

- Thrust T is function of the variation of the axial component of the flow velocity
 - ✓ The flow acceleration due to the enthalpy drop Δh at the propeller
- Once defined the propeller characteristics
 - ✓ Thrust: $T = K_T \rho n^2 d^4$
 - ✓ Propeller's torque: $Q = K_Q \rho n^2 d^5$

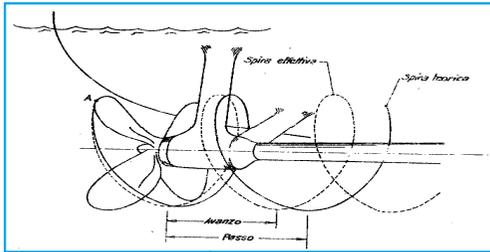
PROPELLER



BOAT

- Thrust T is function of the variation of the axial component of the flow velocity
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Boat dynamic – mathematical model



PROPELLER



BOAT

- $v_a \rightarrow$ relative water velocity behind the hull considering a reference system fixed to the boat
- Compared to the open-water condition the propeller works in a disturbed flow
 - ✓ The water around the aft assumes an advancement motion in the same direction of the ship
 - Wake field
 - ✓ Wake velocity $\rightarrow v_w = v - v_a$
 - $v \rightarrow$ boat speed
 - ✓ Taylor wake factor w
 - $w = \frac{v-v_a}{v} = 1 - \frac{v_a}{v}$
 - $v_w = wv \rightarrow v - v_a = wv \rightarrow v_a = v(1 - w)$
 - ✓ v_a is needed to estimate the dimensionless coefficients of the propeller

Boat dynamic – mathematical model



BOAT

- Newton's second law in longitudinal direction
 - ✓ $T = T_x$
 - ✓ The resistance to the motion R
 - ✓ $m =$ boat mass
 - Acceleration profile
 - Velocity profile

$$\sum F = ma \quad \longrightarrow \quad T - R = ma \quad \longrightarrow \quad v = \frac{1}{m} \int (T - R)$$

Ref.: K. Matveev, Modeling of longitudinal motions of a hydroplane boat, 2012

Ref.: Hydrocomp technical report, Analysis of vessel acceleration with NavCad, 2003

Coupling engine model – boat model

➤ Model with engine governor maps defined into Wave

✓ Maps function of rpm and load

- Fuel mass
- Injection time
- Start of injection

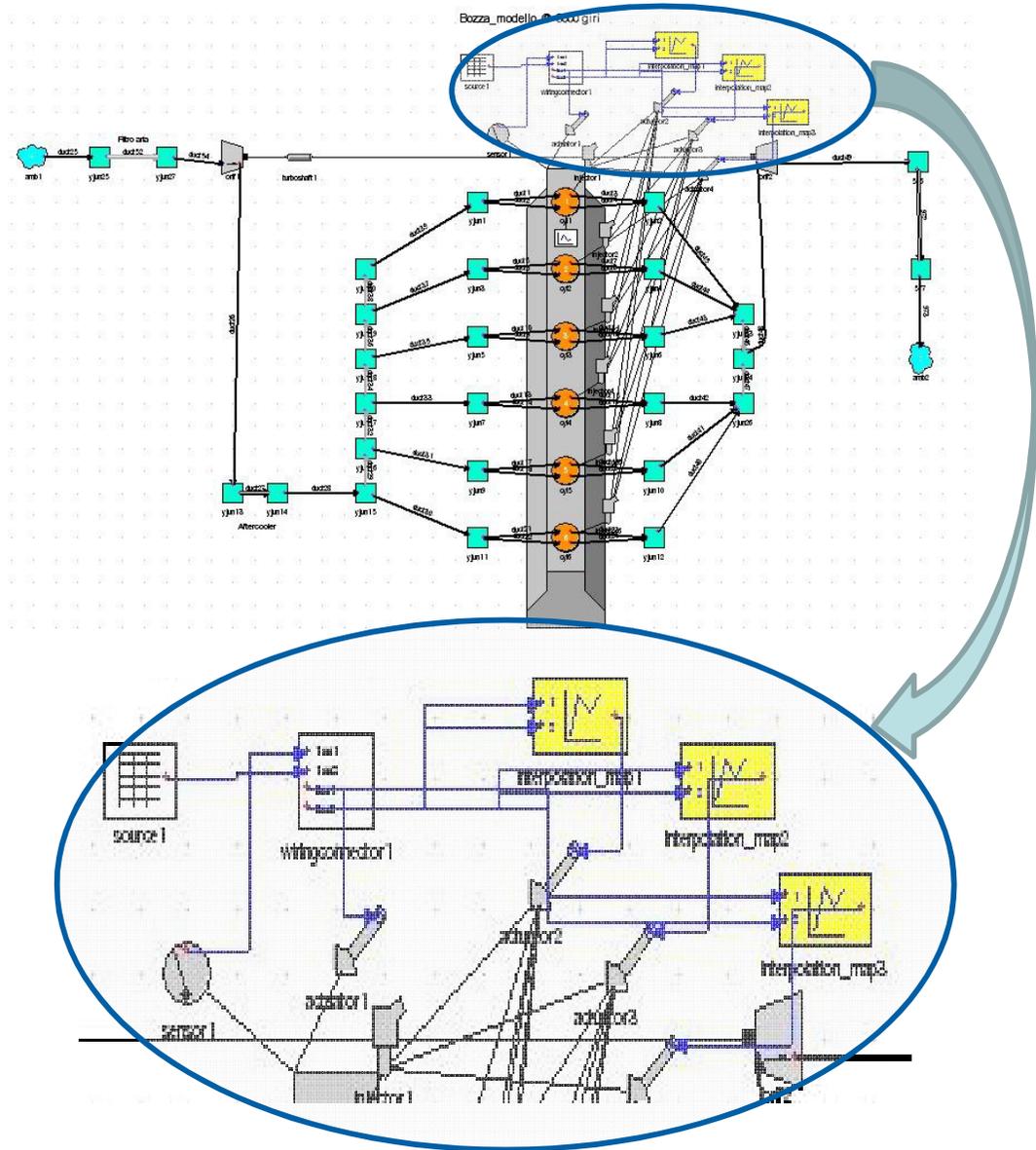
✓ Wave ↔ IGNITE through Wiring-Connector

○ Wave input:

- ✓ Engine rpm
- ✓ Engine load

○ Wave output:

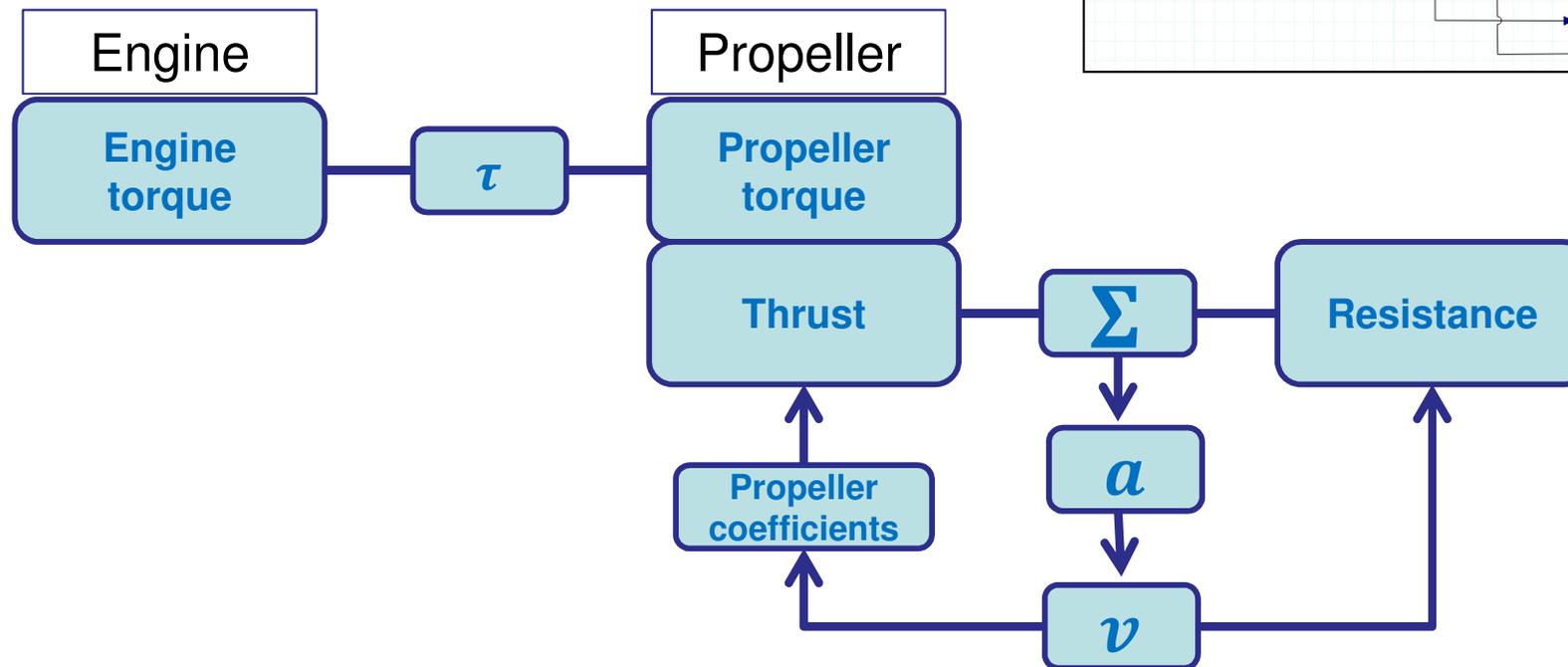
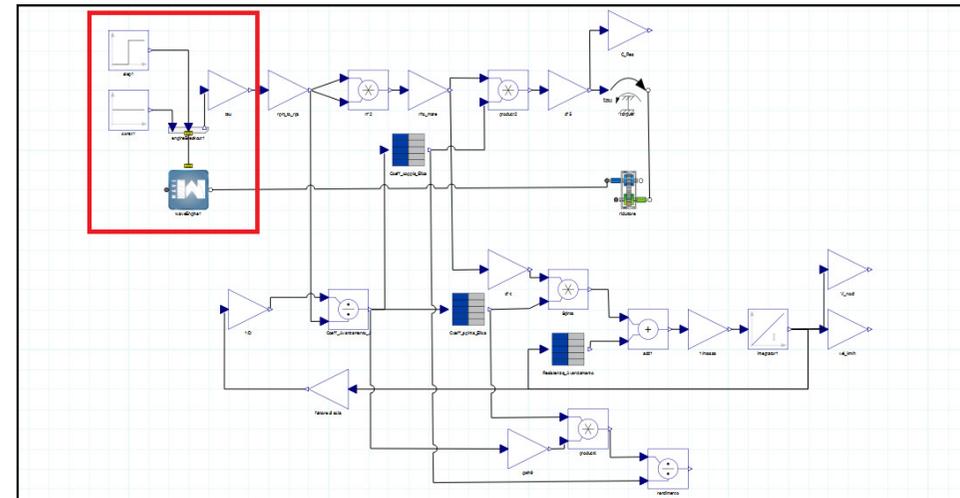
- ✓ Torque [Nm]
- ✓ Consumption [kg/h]
- »



**Further improvement:
Wave RT implementation**

Coupling engine model – boat model

- Engine torque
 - ✓ Load demand profile imposed
 - Step between minimum and maximum load

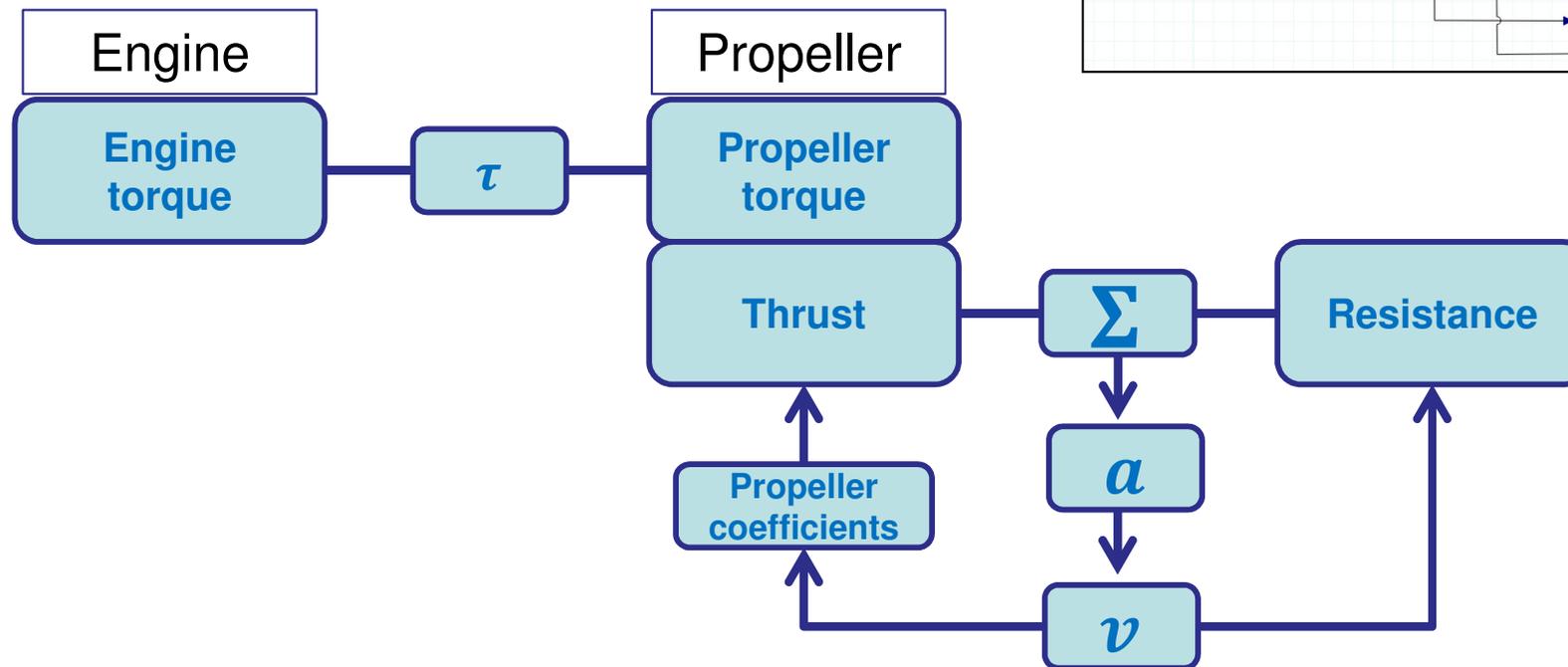
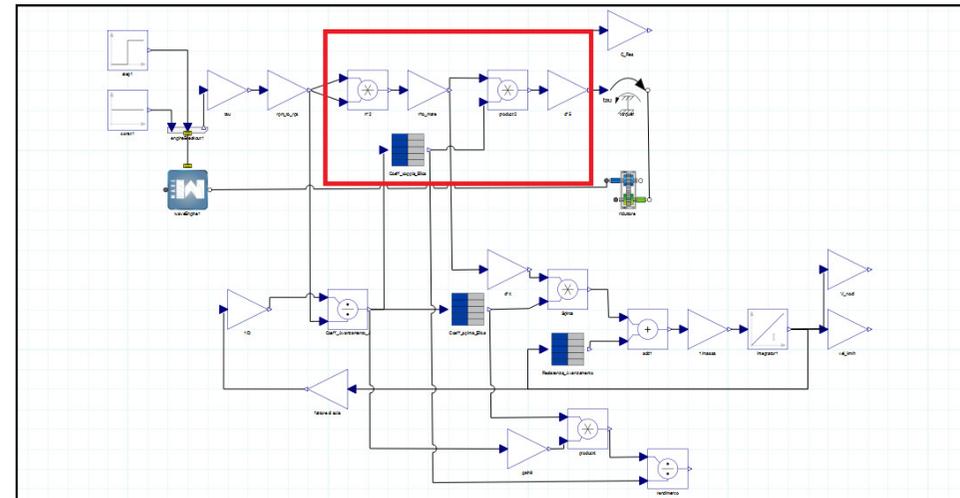


Coupling engine model – boat model

➤ Propeller torque

✓ $Q = K_Q \rho n^2 d^5$

✓ Interpolation table of $K_Q(J)$

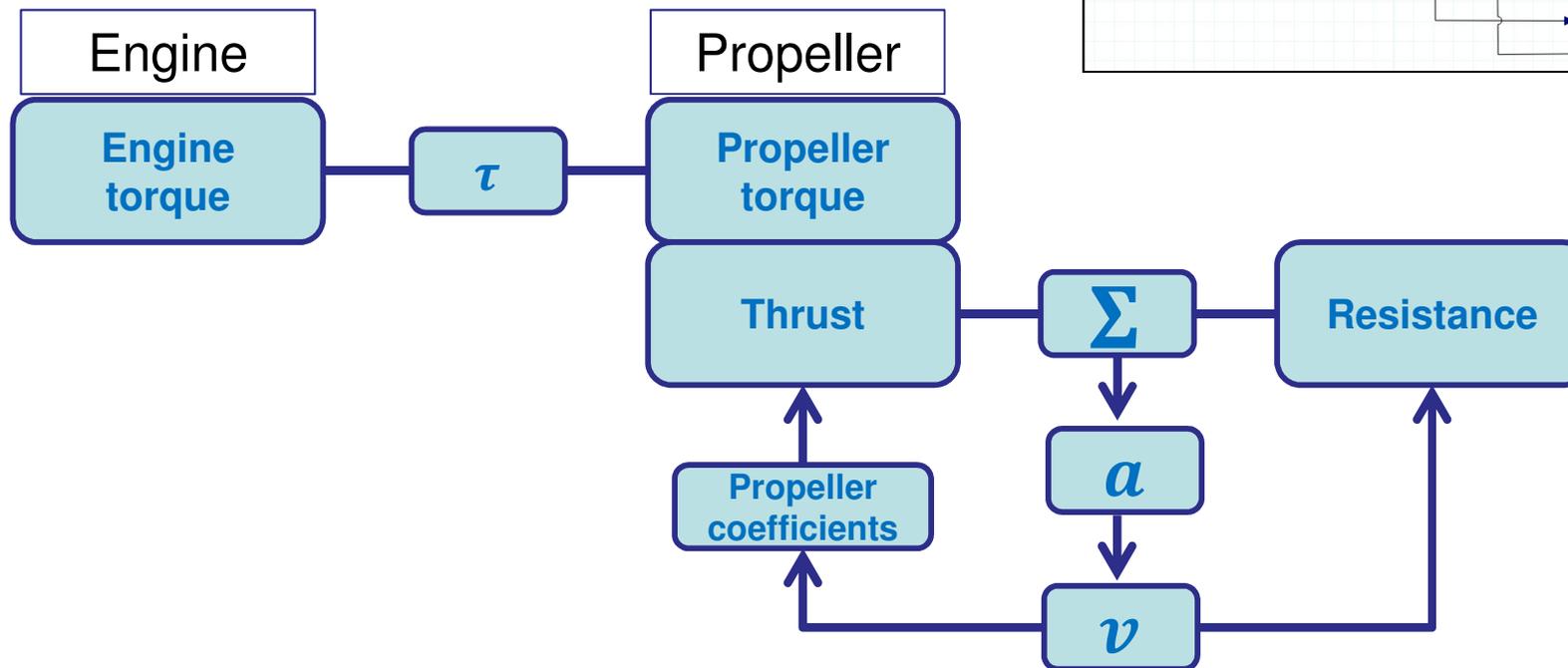
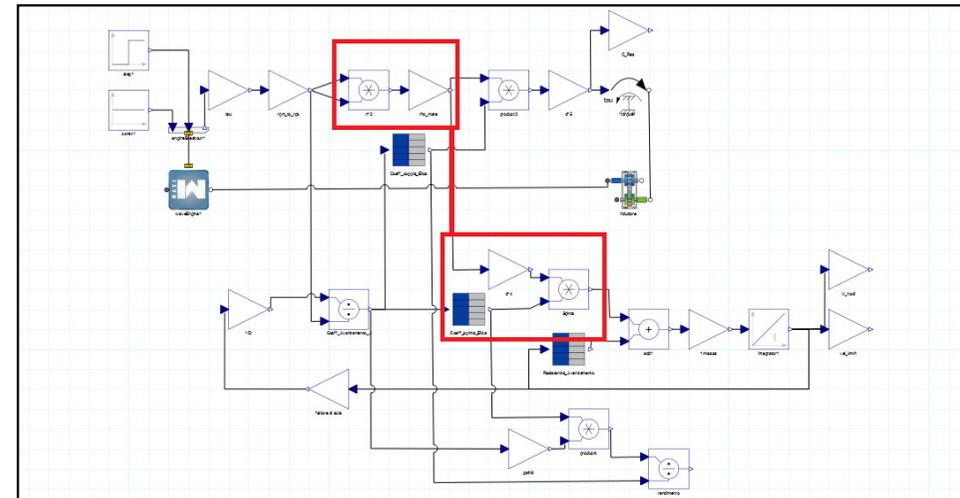


Coupling engine model – boat model

➤ Thrust

✓ $T = K_T \rho n^2 d^4$

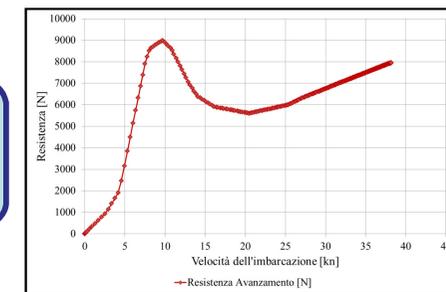
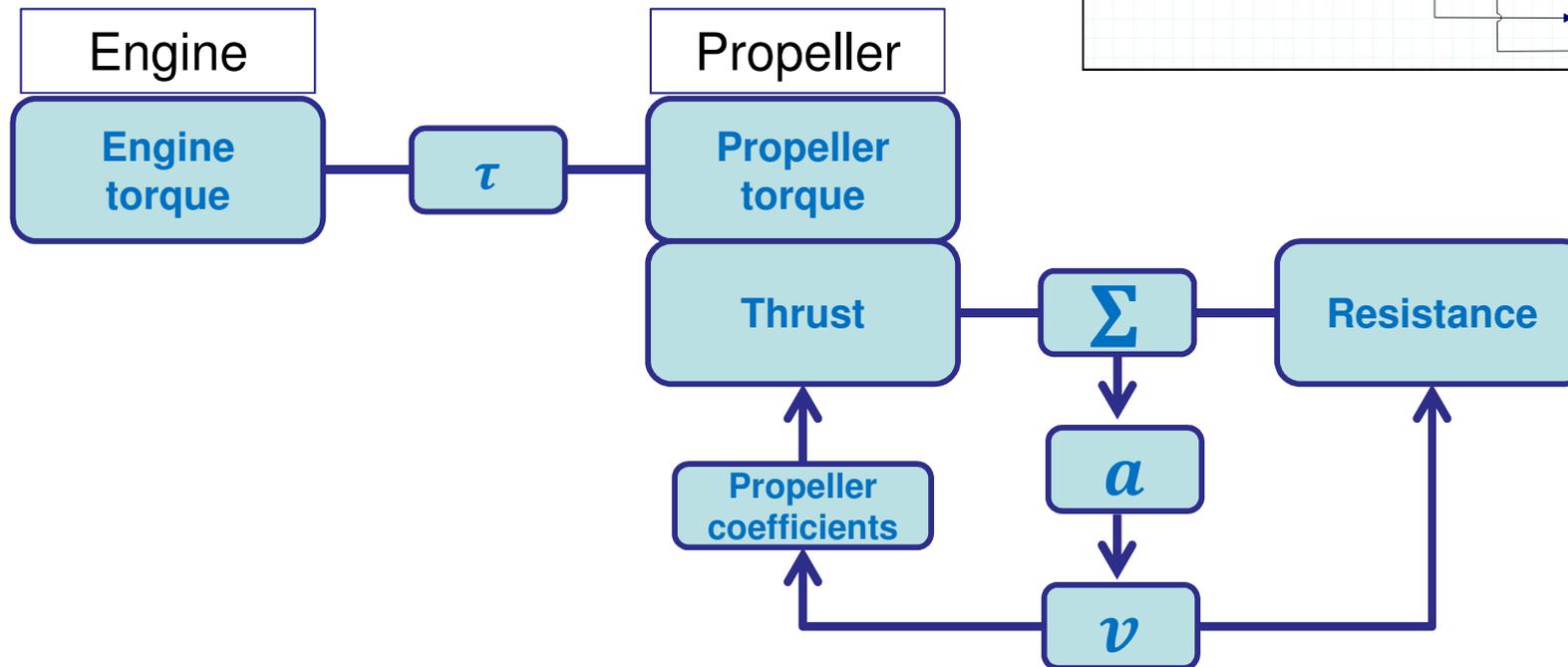
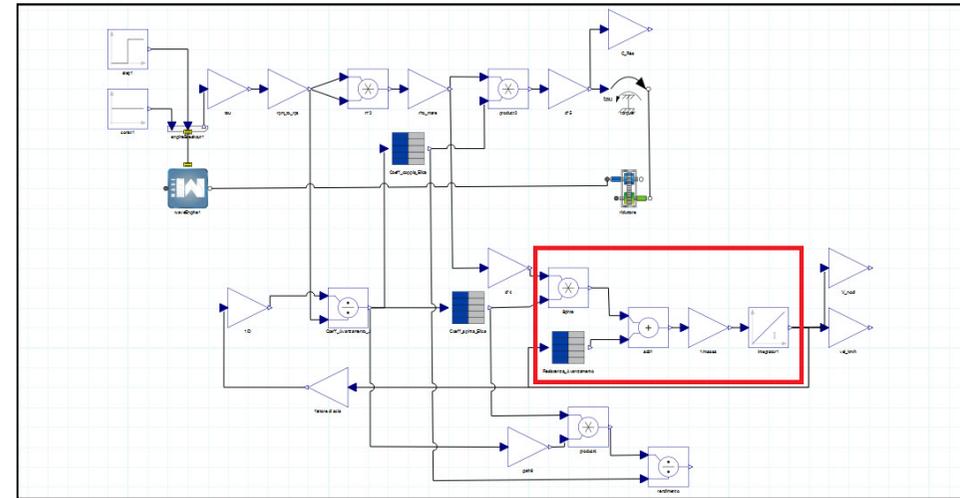
✓ Interpolation table of $K_T(J)$



Coupling engine model – boat model

- Balance between thrust and resistance

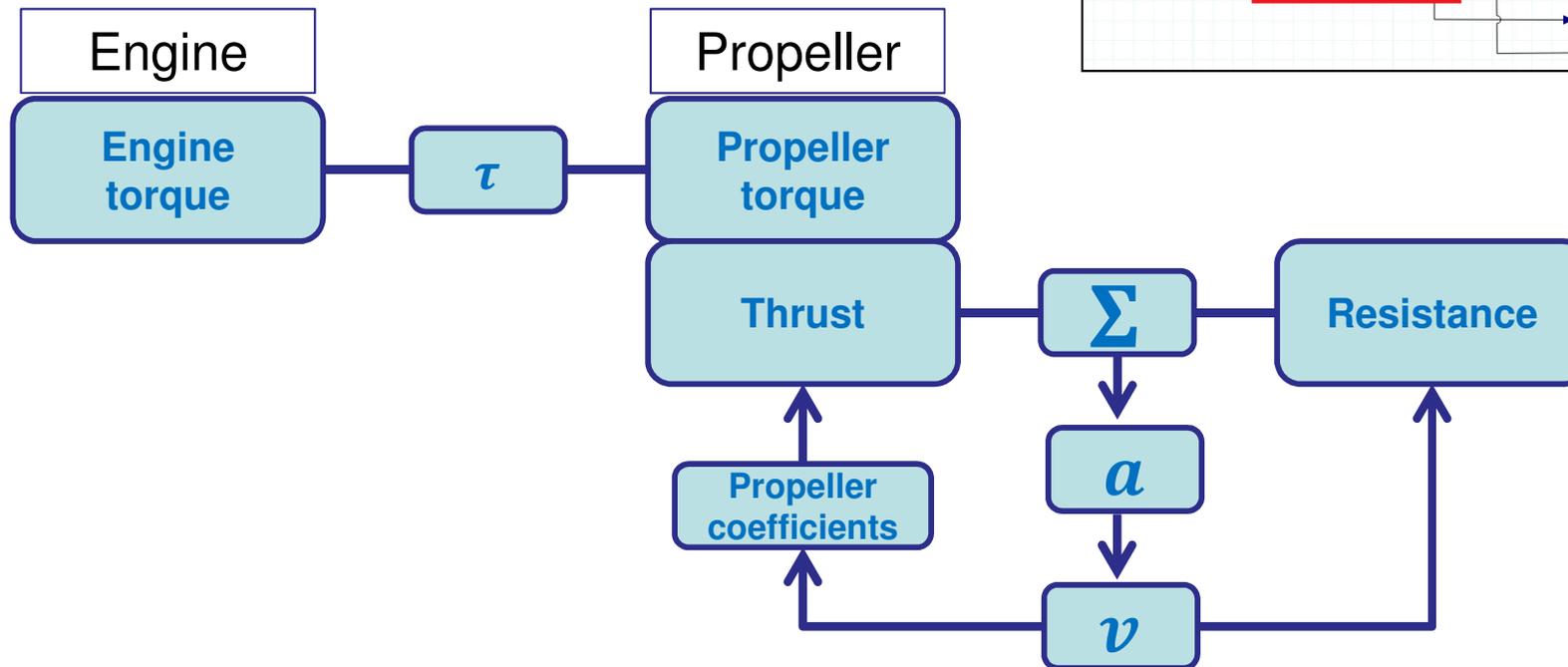
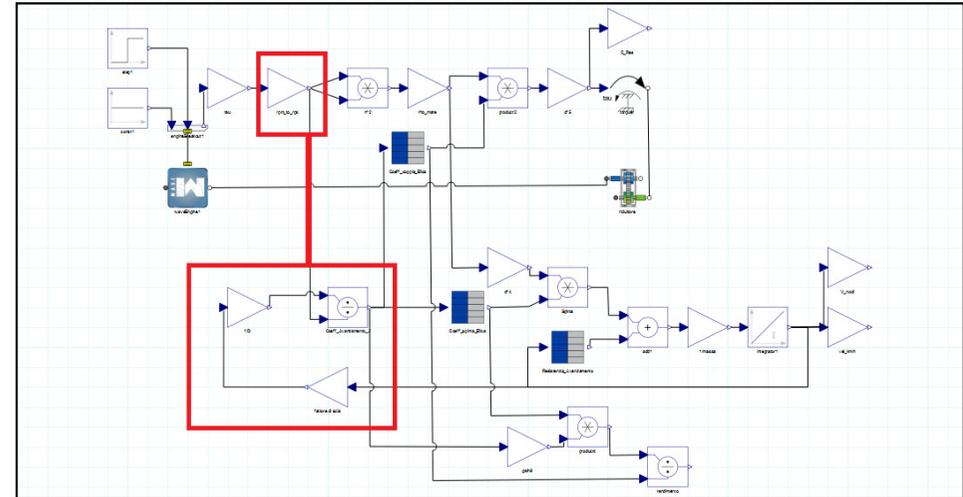
$$v = \frac{1}{m} \int (T - R)$$



Imposed in the model

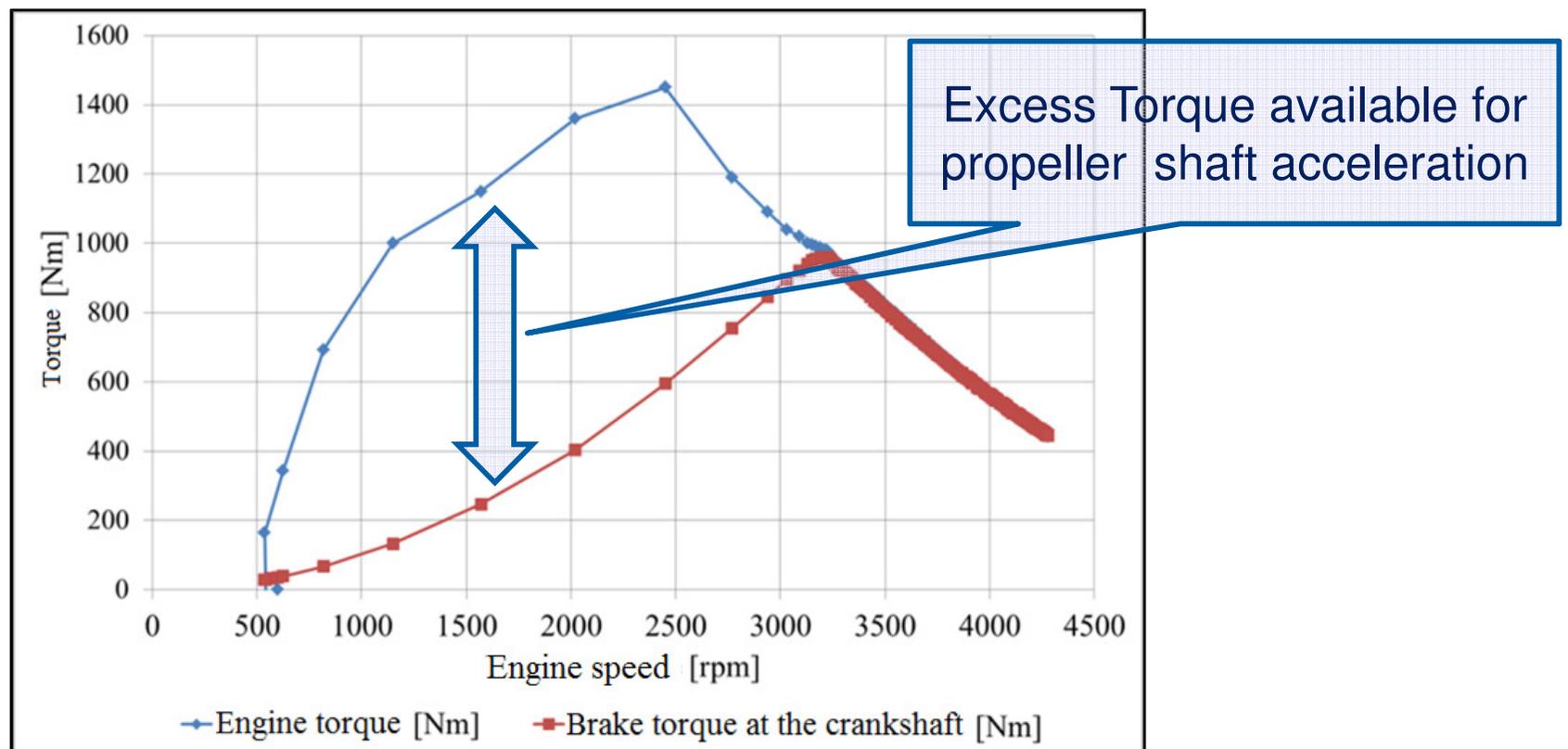
Coupling engine model – boat model

- Velocity v_a
- Advance coefficient J



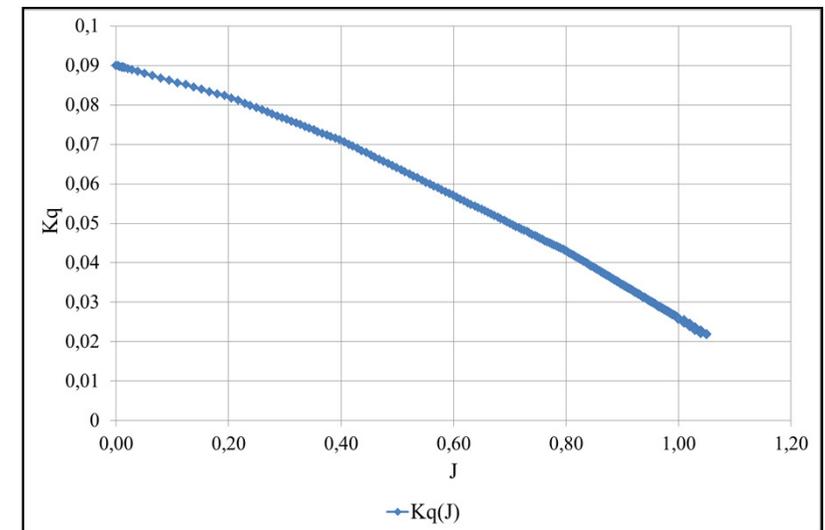
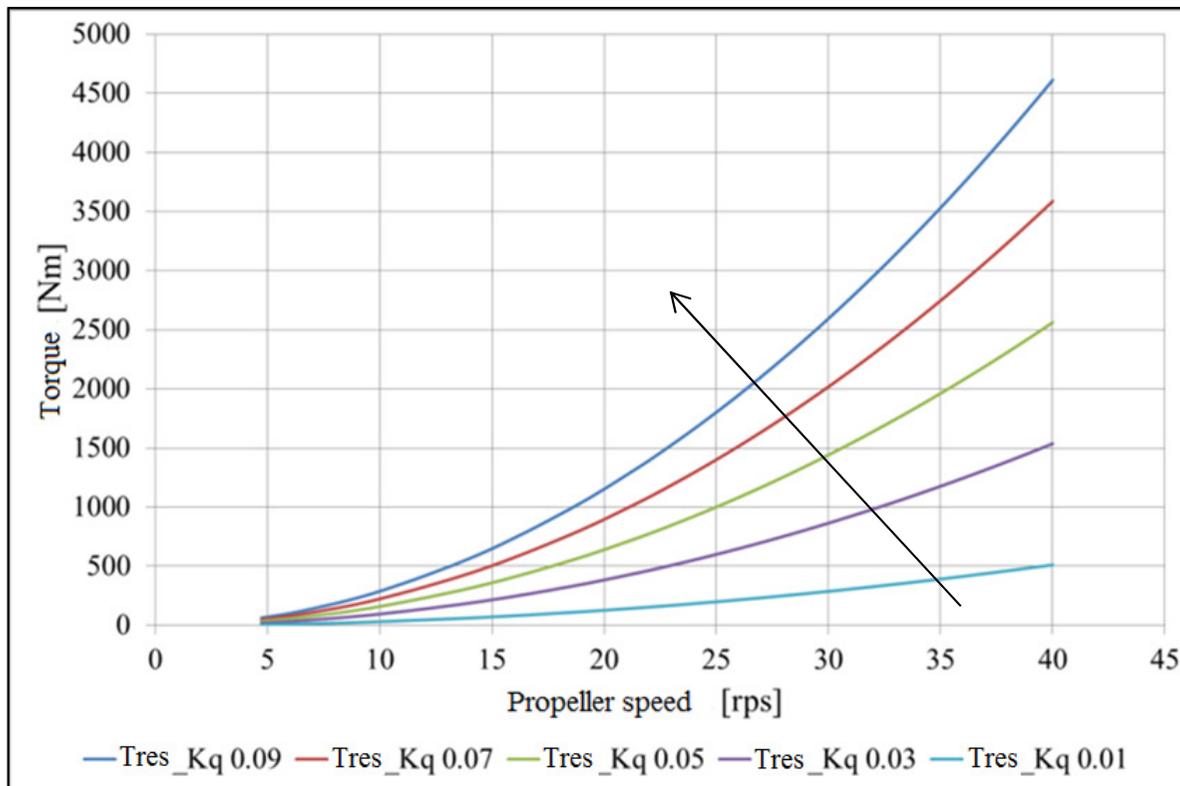
Results

- Engine torque and Brake torque at the crankshaft in function of the engine speed
 - ✓ Imposed an abrupt requested of torque from the minimum to the full load
 - ✓ Analysis of the maximum acceleration of the planing boat



Results

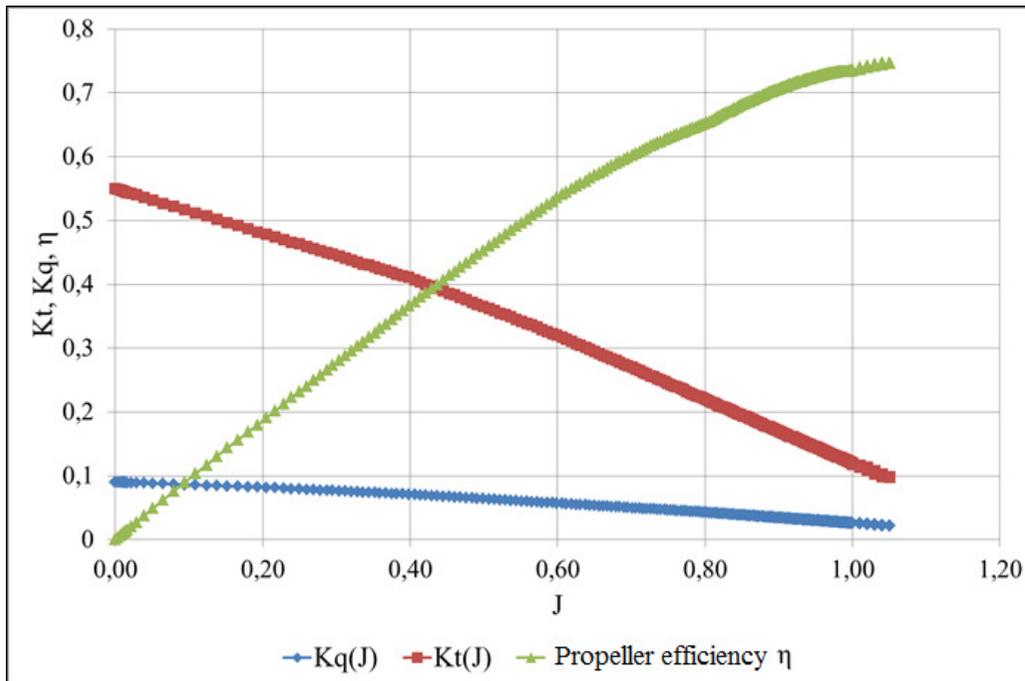
- Brake torque at the propeller shaft $\rightarrow Q = K_Q \rho n^2 d^5$
 - ✓ Graphic in function of the torque coefficient $K_Q(J)$
 - Quadratic trend in function of the rotational speed



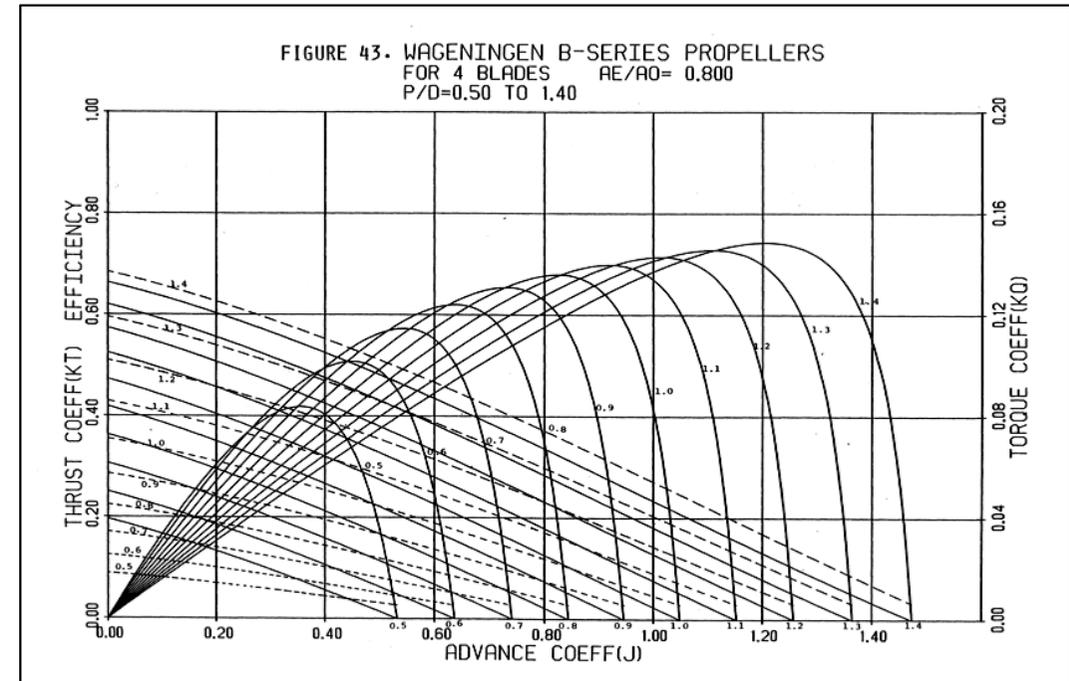
Results

- Trend of the dimensionless coefficients of the propeller

Numerical model

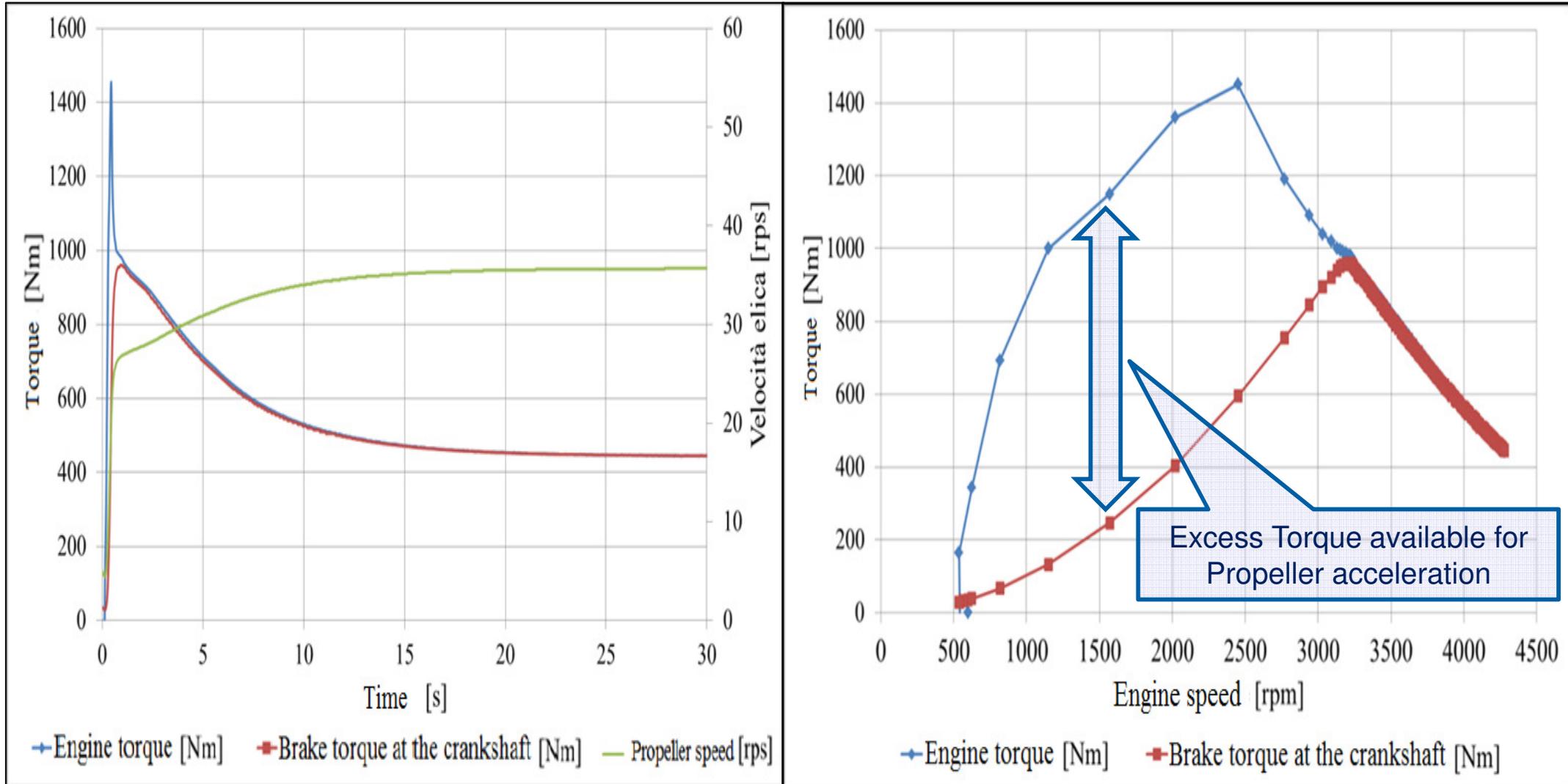


Experimental (from literature)



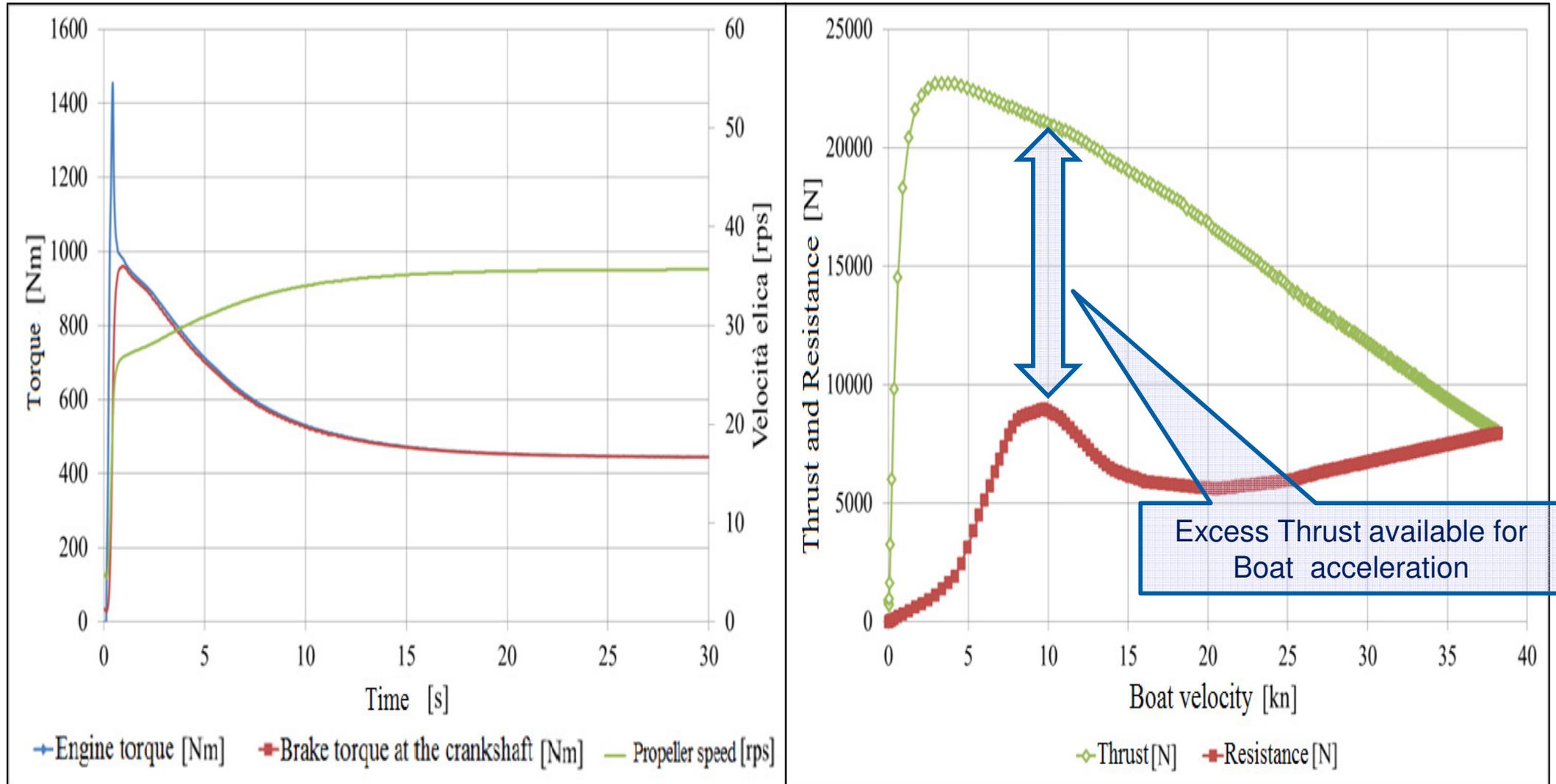
Simulation Results: Transmission dynamics

- Engine torque and Brake torque at the crankshaft in function of time



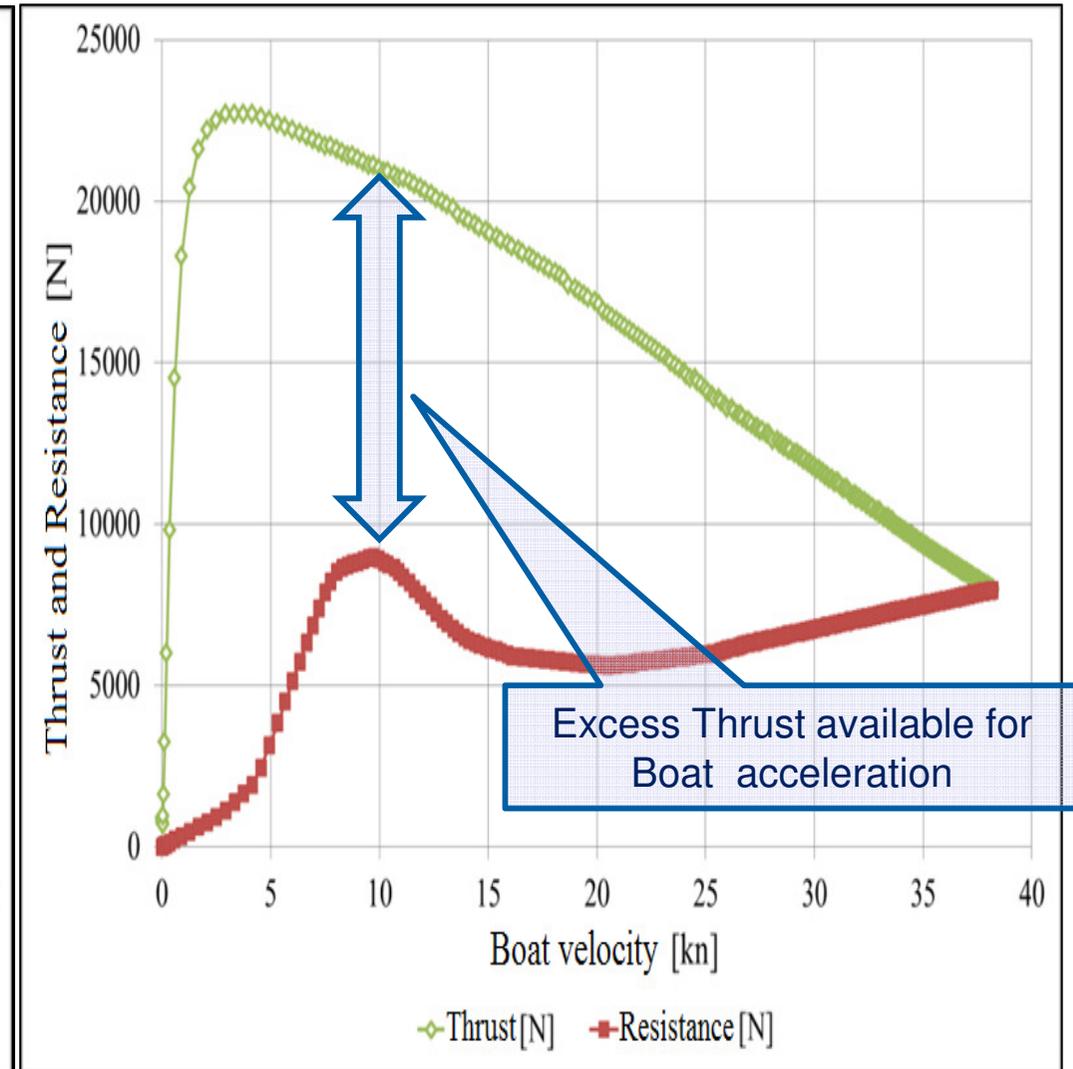
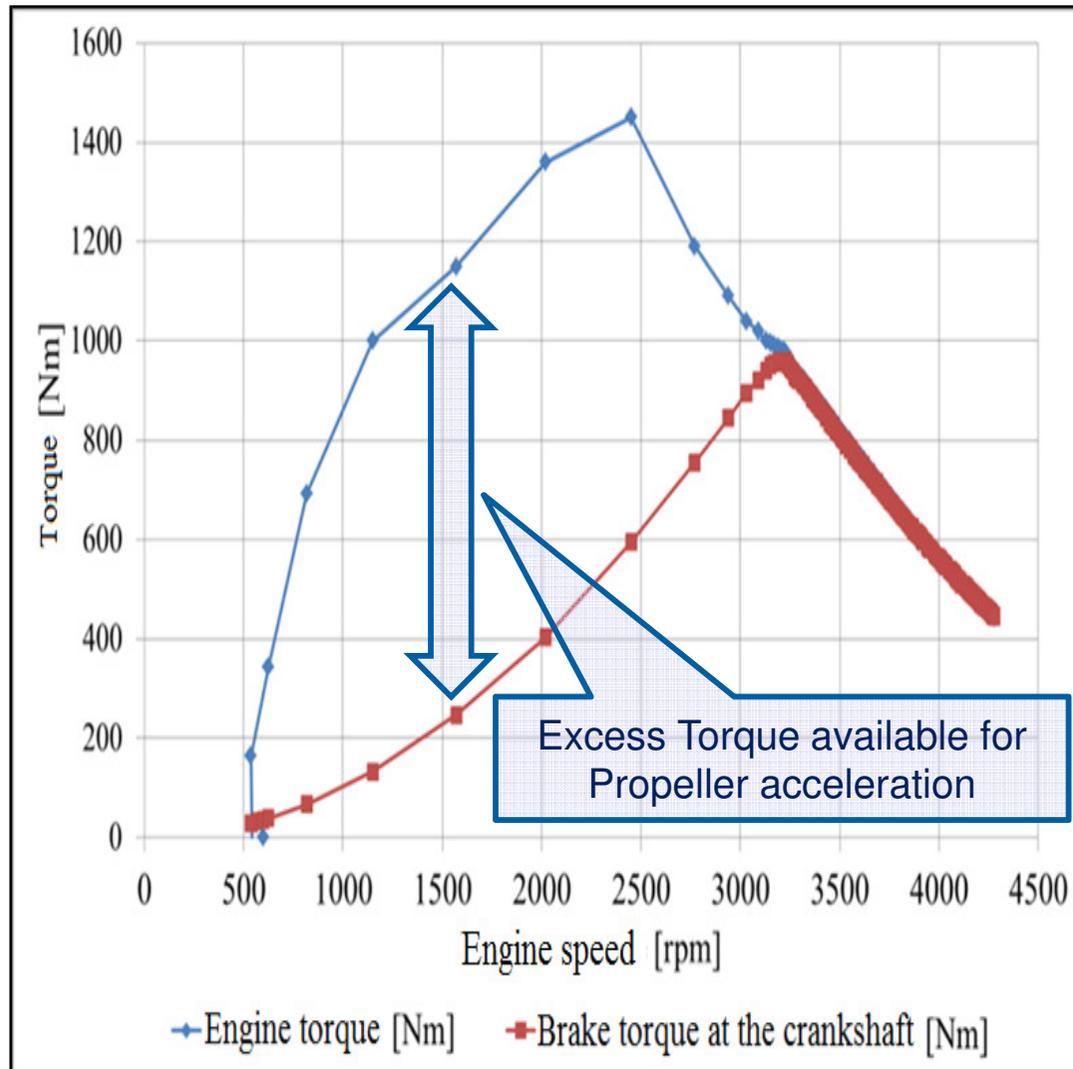
Simulation Results: Transmission dynamics

- Thrust vs Resistance in function of the boat velocity



Simulation Results: Transmission dynamics

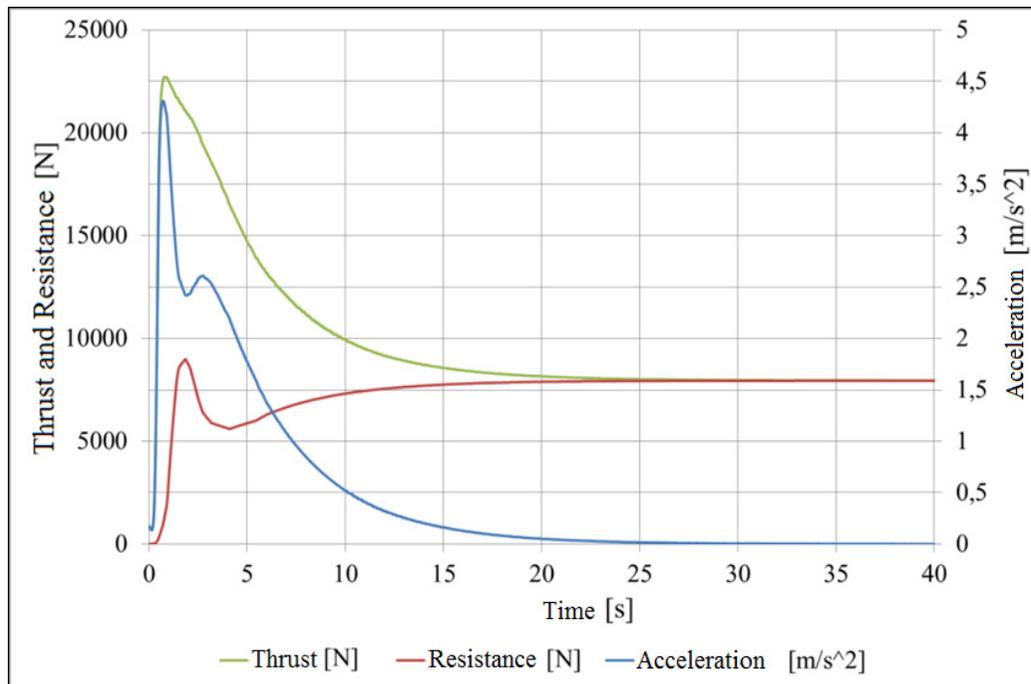
➤ Propeller Thrust vs Engine Torque for boat acceleration



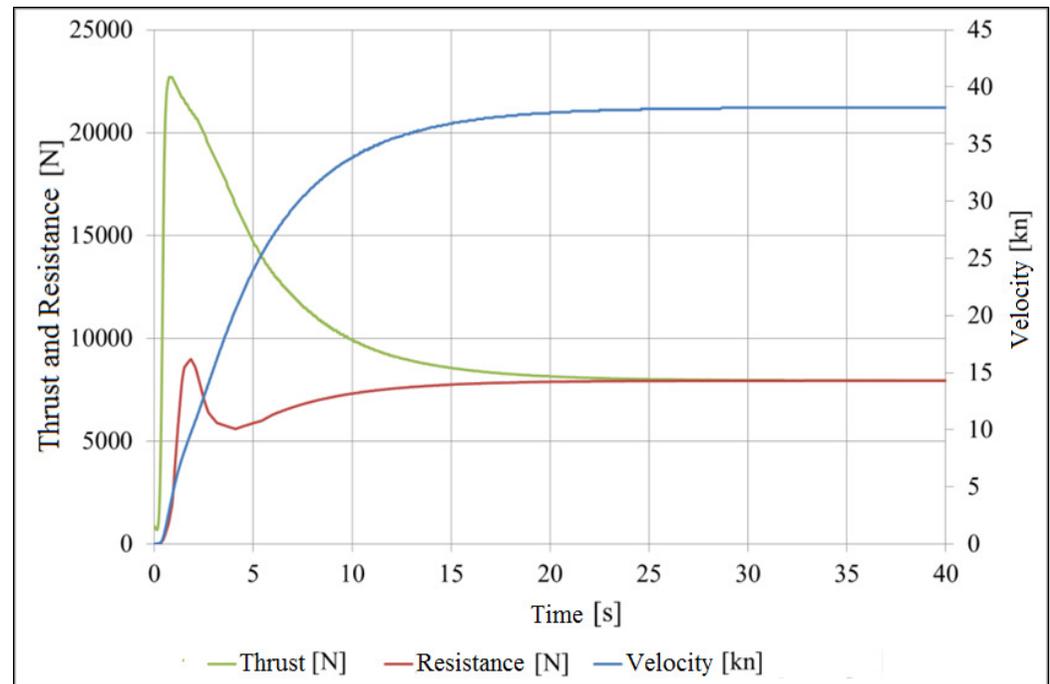
Results

- Thrust – Resistance – Acceleration – Velocity in function of the time

Acceleration

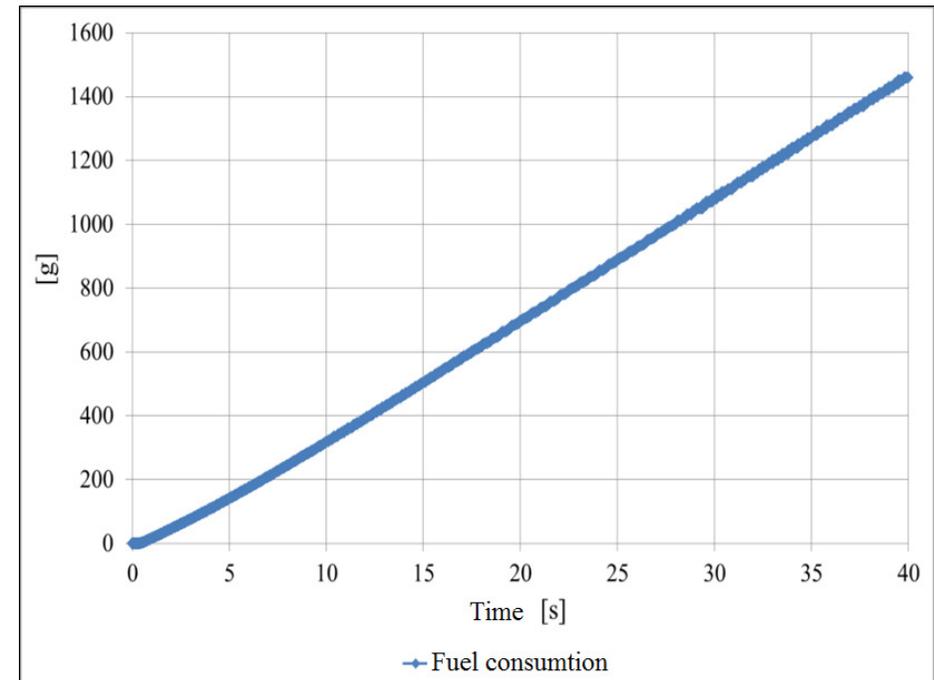
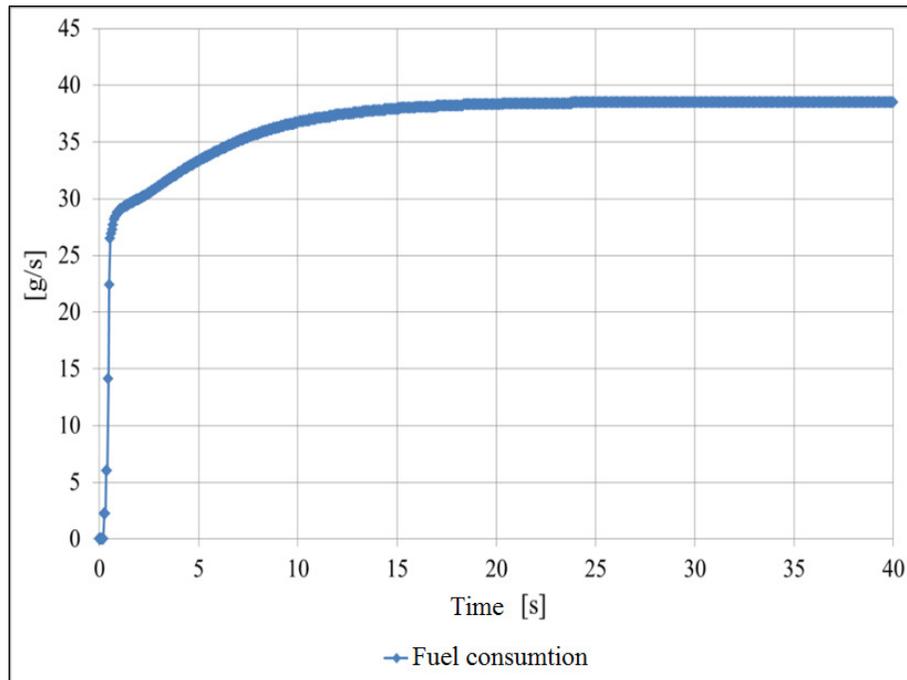


Velocity



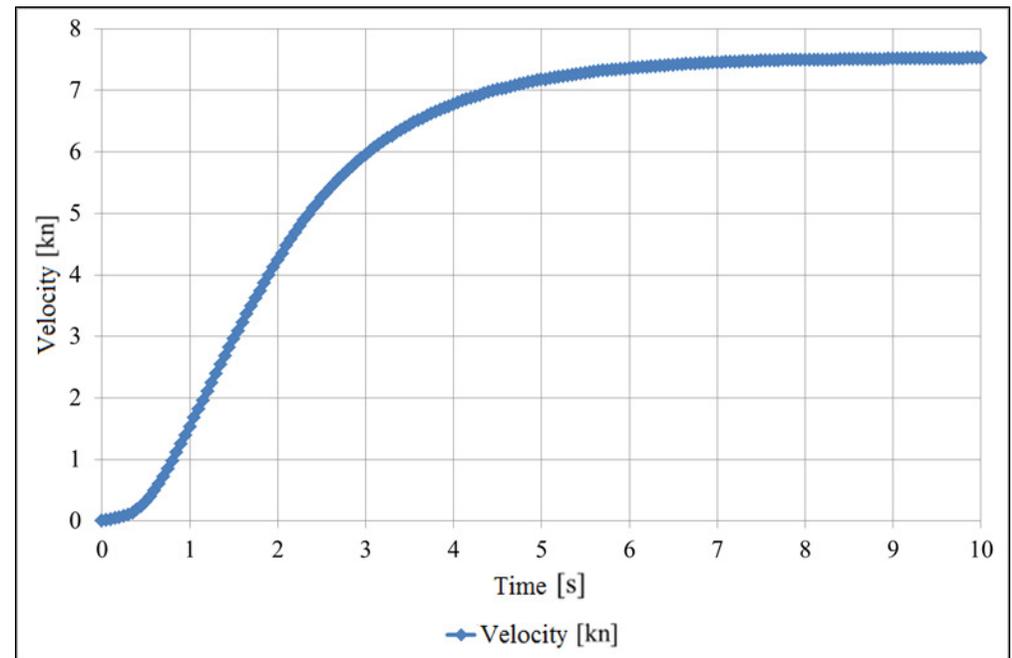
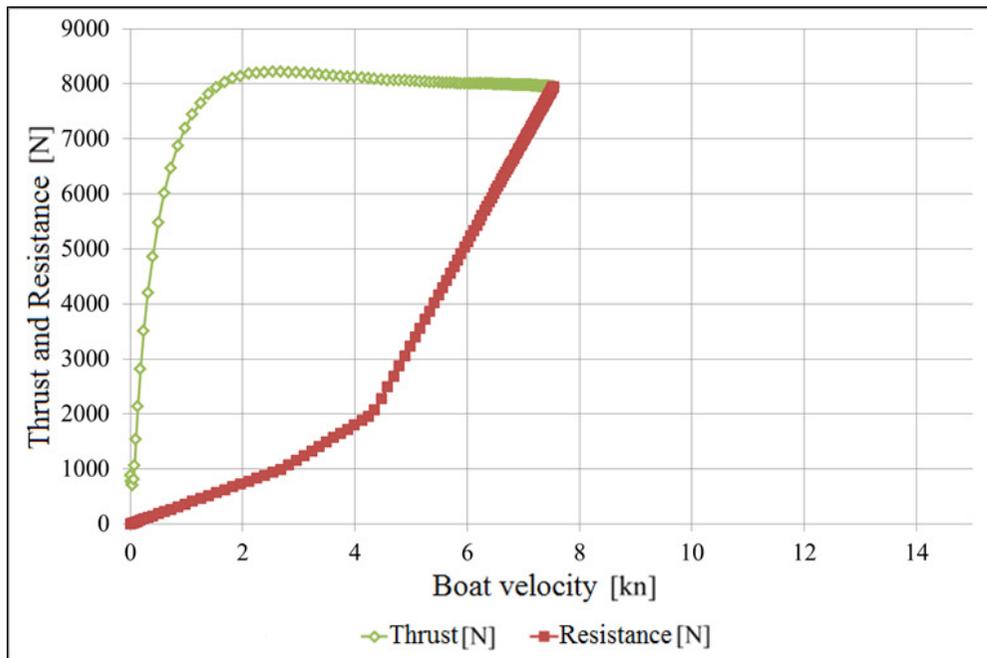
Results

➤ Brake specific fuel consumption



Model Validation in a failure case

- Simulation of the failure to reach the planing phase
 - ✓ Input: reduction of the torque and power of the engine
 - ✓ Result
 - The boat doesn't reach the planing condition
 - Reduction of the maximum boat speed



Conclusions

- First step of development of an integrated engine-boat model for planing hull
 - ✓ Turbocharger engine → analysis of the coupling between engine and turbo
 - ✓ Engine model integrated with a simplified model of the boat longitudinal dynamic
 - Simulation of the boat acceleration and the transition from displacement to planing phase
 - The model's reliability is verified by comparison with literature data

- Future development
 - ✓ Mathematical implementation of the motion resistance equation
 - ✓ Mathematical implementation for the analysis of surface propellers
 - ✓ Coupling with an electrical turbo in order to promote the transition to the planing phase
 - Turbocharger used to recover the surplus energy
 - ✓ Integration with WAVE RT
 - ✓ Experimental validation of the model



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